SPECIFICATION AMENDMENTS

Please rewrite the paragraphs running from line 10 of page 9 to the bottom of page 11 as follows:

A cam ring 12 rotationally couples the coupling jaws 10 to the sleeve 4 and serves as the rear abutment for the spring 16, to which end as shown in FIG. 3 the jaws [[13]] 10 have radially outwardly projecting coupling pins 13 seated in axially and radially open slots 14 of the cam ring 12. At its front end the cam ring 12 has a radially inwardly projecting centering ridge or ring 15 closely surrounding the pusher 8 and keeping it perfectly centered on the axis 6. This ring 12 is internally formed with a front small-diameter cylindrical cam surface 20', a rear large-diameter cylindrical cam surface 20", and a frustoconical cam surface 21 between the surfaces 20' and 20". The jaw front ends 11 bear radially outward on the surfaces [[20,]] 20', 20", and 21 and the diameters of the surfaces 20' and 20" are such that as the coupling jaws 10 are shifted axially from the rear surface 20' over the surface 21 to the front surface [[10',]] 20', the ends 11 are cammed inward to press the teeth 18 of these ends 11 into mesh with the screwthread 19, and axially rearward shifting disengages the teeth 18 from the screwthread 19.

Thus starting from the position of FIG. 1, rotation of the body 2 relative to the sleeve 4 will displace the coupling jaws 10 axially forward. Their front ends 11 will move from a position with their teeth 18 out of mesh with the screwthread 19 to a position meshing with this screwthread 19. More relative forward rotation of the body 2 relative to the sleeve 4 will advance the pusher 8 forward and tighten the clamping jaws 5 on an unillustrated tool shank. Subsequent reverse rotation of the chuck body 2 relative to the sleeve 4 will shift the coupling jaws 10 axially backward until their ends 11 slide down the surface 21 to the surface 20" and leave engagement with the screwthread 19, thereby freeing the pusher 8 to move axially.

The cam ring 12 in FIGS. 1 and 2 is braced rearwardly by a compressible washer [[12]] 22 against the chuck body 2. In FIG. 2 only another spring washer 22' is provided between the front end of the ring 12 and the sleeve 4.

The chuck of FIGS. 5 through 7 has a latching system 23 comprised of an annular row of radially outwardly directed sawteeth 24 formed on the chuck body 2 and a locking member 25 movable by cam formations 26 between an outer position clear of the teeth and an inner position engaged in the teeth 24. The locking member 25 is rotationally coupled to an outer locking sleeve 27 axially nondisplaceable on the sleeve 4 but limitedly rotatable on the sleeve 4. Thus rotation of the locking sleeve 27 in one direction will move the locking member 25 inward, only allowing forward rotation of the chuck body 2 relative to the sleeve 4, with ratcheting of its tip on the sawteeth 24, while preventing any opposite rotation. Thus when locked the chuck can be tightened but will not loosen. Only when the sleeve [[23]] 27 is reverse rotated to allow the tip [[fo]] of the member 25 to leave contact with the teeth 24 can such reverse loosening rotation take place.